

Experimental study on spray and combustion of butanol-diesel blends in a constant volume combustion vessel

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Abstract: Butanol is a very promising alternative fuel and able to be directly used in diesel engines. Visualization studies about spray and combustion of butanol-diesel blends are still scarce. In this work, a constant volume combustion vessel is used to study spray and combustion characteristics of butanol-diesel blends. Ambient oxygen concentration, ambient temperature and butanol concentration range from 9% to 21%, from 800 K to 1200 K and from 0% to 20%, respectively. Results show that the ascent stage of the FINL under 21% approximately coincides with that under 15% at 1200 K. The FLOL of 21% is approximately equal with that of 15% at 1200 K. The TINL of oxygen concentration 15% is the greatest at 1200 K. However, the TINL of 21% is higher than that of 15% at 800 K. These results can become important references for the choice of working conditions (EGR rate, injection timing and butanol concentration).

Introduction

Butanol, a kind of renewable fuels, has drawn a lot of attentions in recent years due to limited fossil fuel resources on earth. Because oxygen content in butanol and excellent miscibility with diesel, butanol is a promising alternative fuel in diesel engine.

Wu et al. [1-4] utilized a constant volume combustion vessel to study the effect of different ABE proportions on spray and combustion. Results indicated that addition of ABE prolonged the ignition delay and the flame lift-off length (FLOL), but reduced the peak of space integrated natural luminosity (SINL) and time integrated natural luminosity (TINL). Butanol has several advantages over ethanol including higher energy density, more excellent miscibility with diesel and much less hygroscopic. Compared to diesel, butanol is featured by lower viscosity, lower surface tension, higher latent heat and lower cetane number. These features of butanol are conducive to enhancing atomization and prolonging the ignition delay. So far, the majority of studies on butanol-diesel blends have been conducted on engine benches. Fundamental characteristics of spray and combustion under various conditions, which are not easily obtained through engine bench tests, are still deficient. Based on this situation, we investigated the impact of different conditions on basic spray and combustion characteristics of butanol-diesel blends through a constant volume combustion vessel.

Experimental facilities and conditions

The constant volume combustion vessel with a cubic inner length of 134 mm was used in this study (see Fig. 1). The vessel body was heated to 383 K to prevent water condensation on optical

windows. The schlieren system with a halogen lamp and a pair of 150 mm paraboloid mirrors was used to visualize the development of spray and combustion. The high speed digital camera (IDT MotionPro Y4-S1), which was set to 20000fps with a resolution of 640×280 pixels, was used to record images of the spray and combustion process. By utilizing current optical setup, the spatial resolution of images was 0.204 mm/pixel. From Fig. 1, pressure profiles of combustion processes are measured by a piezoelectric pressure transducer (Kistler 6052C), a charge amplifier (Kistler 5018) and a DAQ card (NI USB-6251M).

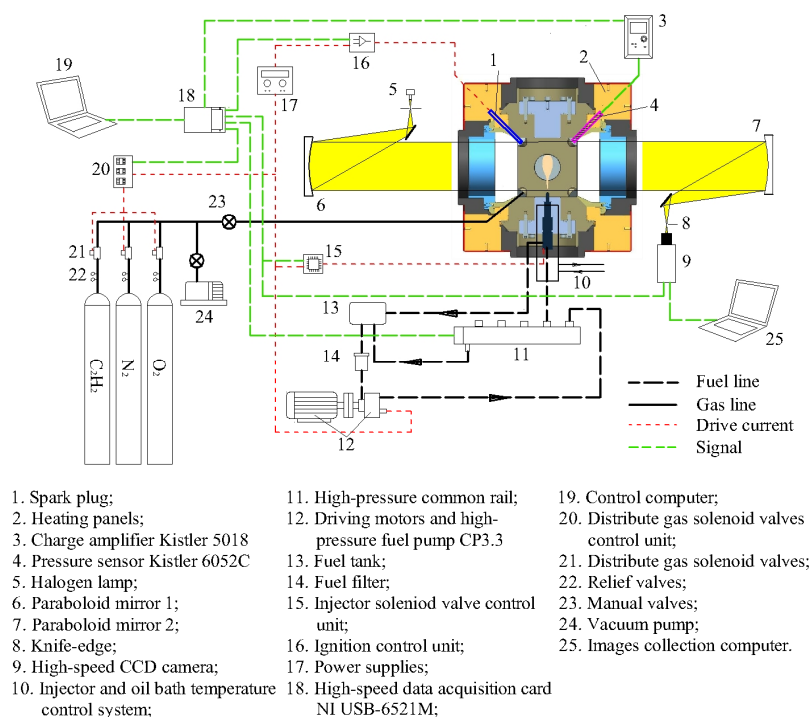


Fig. 1 Schematic of constant volume combustion vessel[5]

In order to avoid over-exposure and observe the flame luminosity distribution, two ND 8 filters were placed in front of a 100 mm focal length lens (Tokina 100 mm F2.8 macro). Two ND8 filters reduced the brightness to its 1/8 each and 1/64 totally[5]. During every test process, the camera and pressure acquisition system began recording after triggered by a TTL falling edge signals synchronized with injection signal [5]. Besides, the time interval between the start of injection (SOI) and injection signal should be obtained through the high speed digital camera. The SOI was regarded as the original point of flame images and pressure datum.

The start of combustion (SOC) was defined as the timing when 15% of total heat release is reached[3]. Thus, ignition delay was determined as the period between the SOC and the SOI[2]. In order to eliminate environment disturbances, baseline background, the frame right before SOI, was subtracted from flame images [5]. In addition, the threshold was set to 15% of the maximum pixel value to avoid noise [2]. The SINL was calculated by integrating the pixel values over the entire image[3]. The TINL was calculated by further integrating the SINL with time[3].

The experiment is conducted with three different butanol concentrations—B00, B10, B20 (B20 represents the blend of 20% (mas.) butanol and 80% (mas.) diesel). Main reasons for the selection of

butanol proportions are as follows: (1). If butanol concentration is too low, the influence of butanol on spray and combustion can't be observed; (2). If butanol proportion is in a large amount, it makes combustion unstable especially under cold start condition; (3). Higher ratio of butanol ($\geq 20\%$) contributes to precursors of PAHs such as toluene and benzene [6]. Different ambient oxygen concentrations (21%, 15% and 9%) and ambient temperatures (800K, 1000K and 1200K) correspond to different EGR rates and injection timings, respectively. If the early injection strategy is adopted, the start of combustion (SOC) is closely related to ignition delay.

Results and discussion

The natural luminosity results from two sources: chemiluminescence and soot incandescence. The natural luminosity of soot incandescence is much stronger than that of chemiluminescence [3]. After using ND filters, chemiluminescence of radicals such as OH is filtered almost entirely [5]. Thus it's reasonable to reckon that natural luminosity stands for soot incandescence. Furthermore, the soot luminosity is related to two parameters: soot concentration and soot temperature [4]. Soot formation mainly depends on local flame temperature and local equivalence ratio.

Because spray images for different fuels under various conditions are very similar in terms of flame structure, only images of B10 are shown in Fig. 2. The orifice of the injector is located on the left side of images. Raw pictures acquired by the high speed camera are gray images, which are not easily recognized by eyes. Thus the pseudo-color maps are created by matlab, with yellow and red for flame and black for background. It's obviously observed that the natural flame luminosity at 800 K and 9% is barely detectable after using two ND filters. Jangi et al. [7] made use of numerical simulation to study the effect of oxygen concentrations on combustion, and indicated that the highest flame temperature at 1000K and 10% was only 1980K. Lower ambient temperature and oxygen concentration are likely to reduce flame temperature further. Soot can't be created if the flame temperature is below 1400K-1600K [8]. Therefore, it's reasonably believed that little soot is produced at the condition of 800K and 9%.

For all tested cases, there is a stable region without natural luminosity between the orifice and flame before 2.5 ms. Afterwards, flame luminosity rapidly spreads towards the orifice at 3.0 ms and fiercely propagates to the right side of the image like wavelike structures after 3.0 ms owing to the weakened spray momentum. This speed of flame propagation after injection becomes progressively faster with the increase of oxygen concentration, because high oxygen content enters into the interior of spray and accelerates soot oxidation. The rules of change about flame also are presented in Fig. 3 (a) and (b).

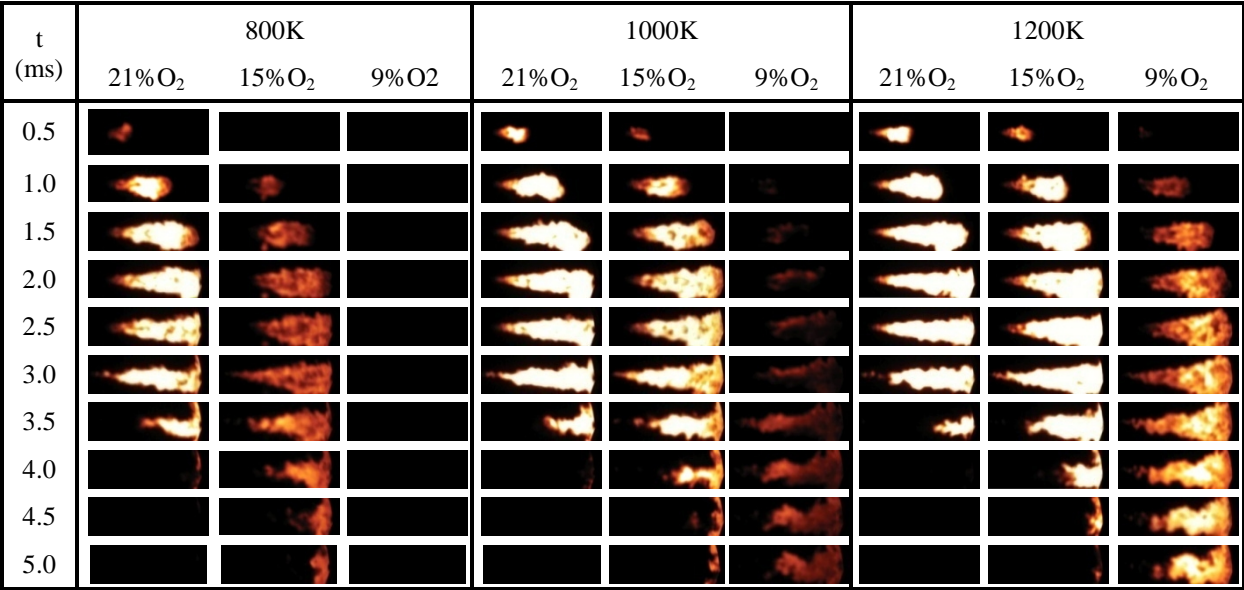


Fig. 2 Natural flame luminosity of B10.

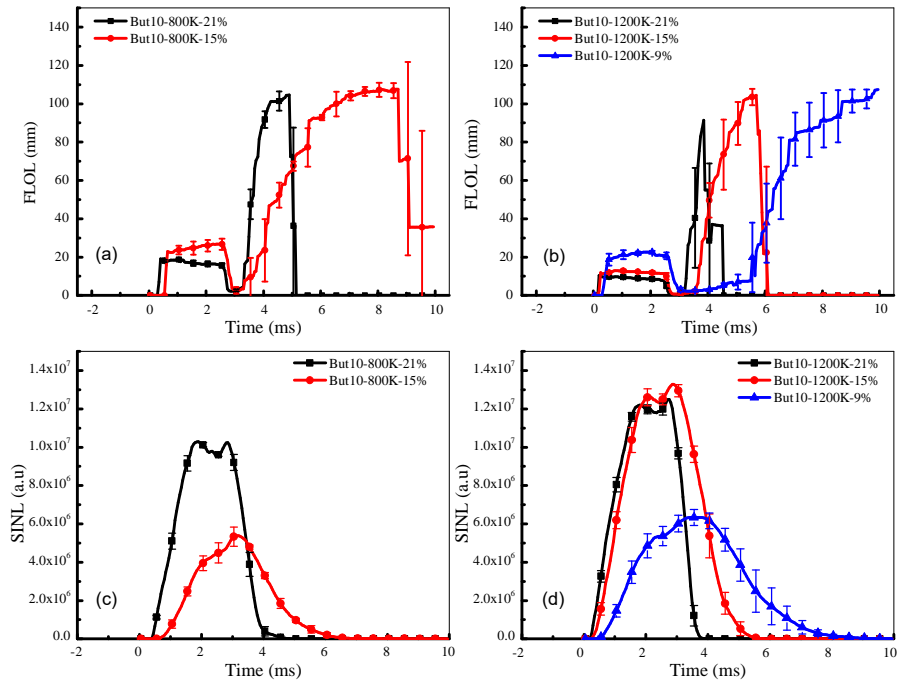


Fig. 3 The effect of oxygen concentration on FLOL and SINL

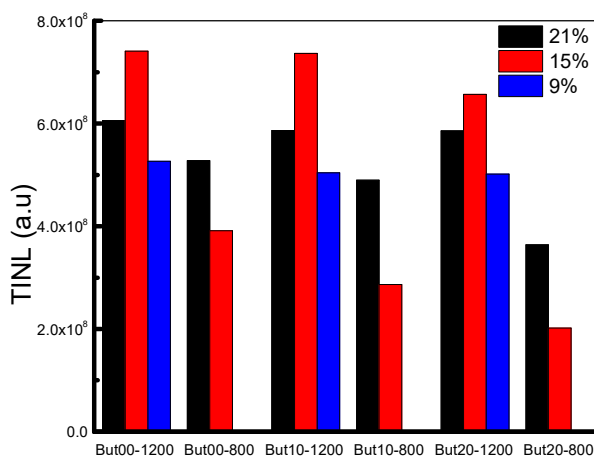


Fig. 4 The effect of oxygen concentration on TINL

As shown in Fig. 4, the TINL of 15% is the largest at 1200 K, while the TINL of 21% is larger than that of 15% at 800 K. At the ambient temperature of 1200 K, flame luminosity and structure of 15% are extremely similar to those of 21% before 3.0 ms (see Fig. 2). From Fig. 3 (d), one also observes that the SINL curve of 15% and 21% almost coincide with each other before 3.0 ms. Generally, the overall ignition delay of the spray combustion consists of two parts: physical delay and chemical delay. Since the ambient oxygen concentration has little impact on liquid penetration length [3], the oxygen concentration which doesn't work on physical delay plays a role in chemical delay. In spite of the dilution effect due to the decrease of oxygen concentration (from 21% to 15%), the ignition delays and FLOL of 15% and 21% both are very short because high ambient temperature (1200 K) is the dominant factor. As stated earlier, flame disappears faster with the increase of oxygen concentration, therefore the TINL of 15% is more than that of 21% at 1200 K. Because the ambient temperature no longer is dominant with the decrease of ambient temperature, the dilution effect of low oxygen concentration gets more and more important. Therefore, FLOL increase with the decrease of oxygen concentration at 800 K (see Fig. 3 (a)). The larger gap between liquid penetration length and FLOL allows more air entrained into the spray region, which result in the decrease of SINL (see Fig. 3 (c)). Finally, TINL gradually decrease with the decrease of oxygen concentration at 800 K.

Conclusion

A constant volume combustion vessel was used to study spray and combustion of butanol-diesel blends under different ambient condition. The following major conclusions were drawn from this study.

- (1). At the ambient temperature of 1200 K, the ascent stage of the FINL under 21% approximately coincides with that under 15%. The FLOL of 21% is approximately equal with that of 15%.
- (2). The TINL of 15% was the largest under 1200K, while the TINL of 21% was larger than that of 15% under 800K.

Acknowledgement

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